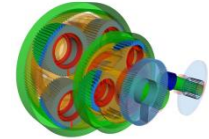


SIMULIA 3DEXPERIENCE Conference
Design, Modeling and Simulation



OTraPArTe: Order Analysis and Identification of Critical Transfer Paths in Drive Trains

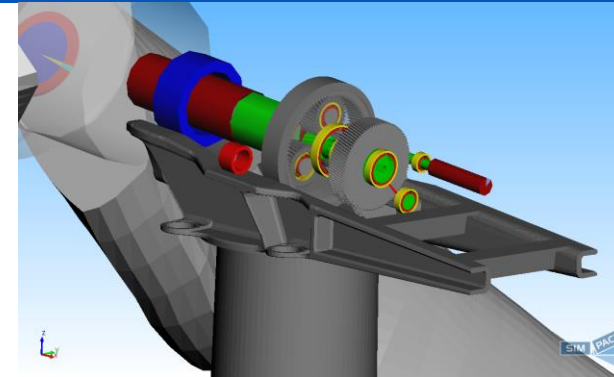
Darmstadt, Germany, 20.11.2019
Marko Grandy - Stefan Hauptmann

marko.grandy@mesh-engineering.de wind@mesh-engineering.de



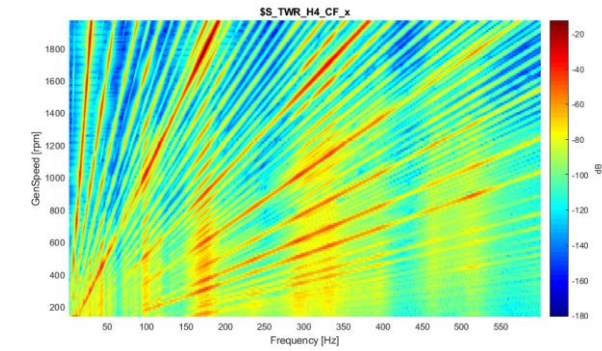
1. Introduction - Vibration in rotating machinery

- What are the problems?
- Where do they come from?



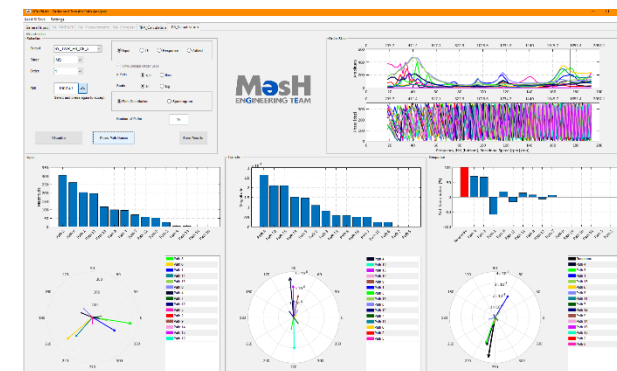
2. Identification of vibrations and their sources

- Order Analysis with OTraParTe



3. Understanding the transfer path of a vibration

- Transfer Path Analysis with OTraParTe



4. Conclusion

1 Introduction – Vibrations in rotating machinery

Speed-dependant excitations

- Gears
- Motors
- Generators

Transmission via all connected components

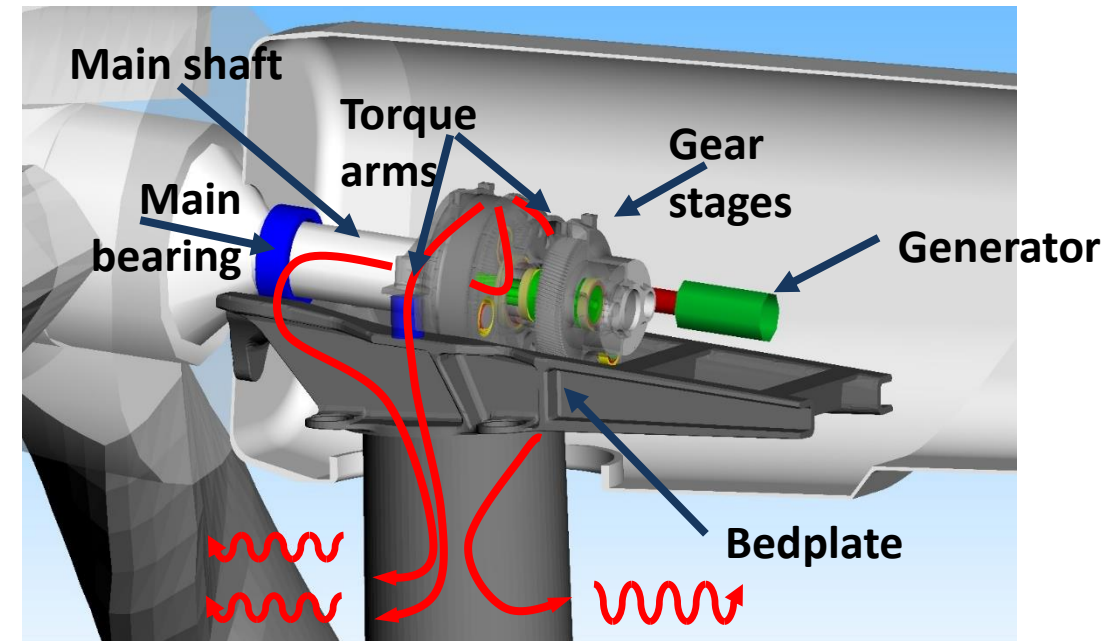
- Shafts
- Bearings
- Support structure

Dynamic loads

- Fatigue

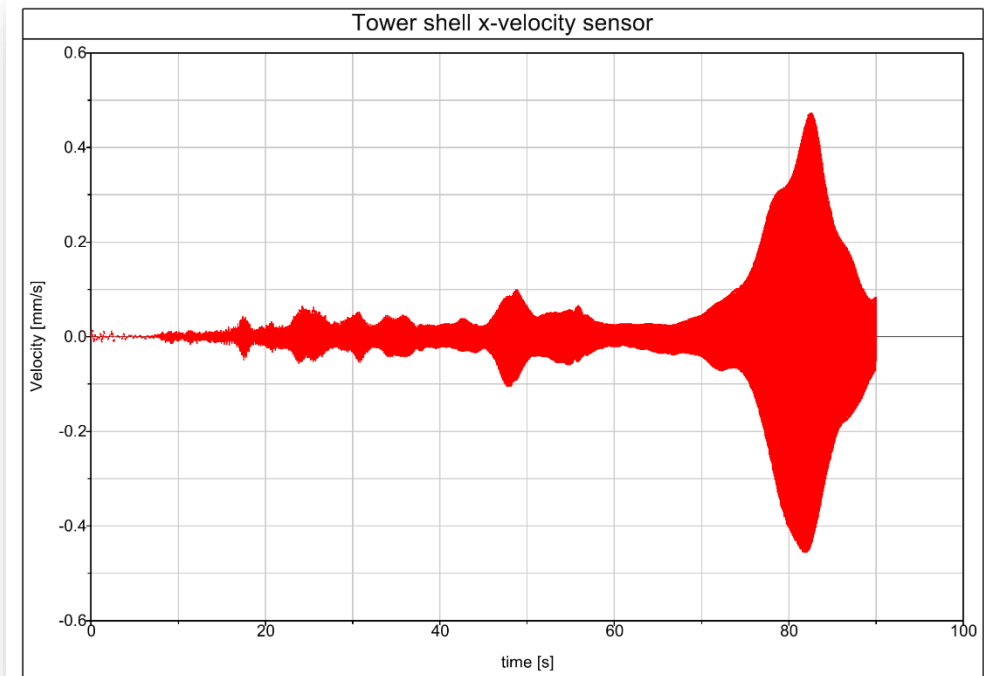
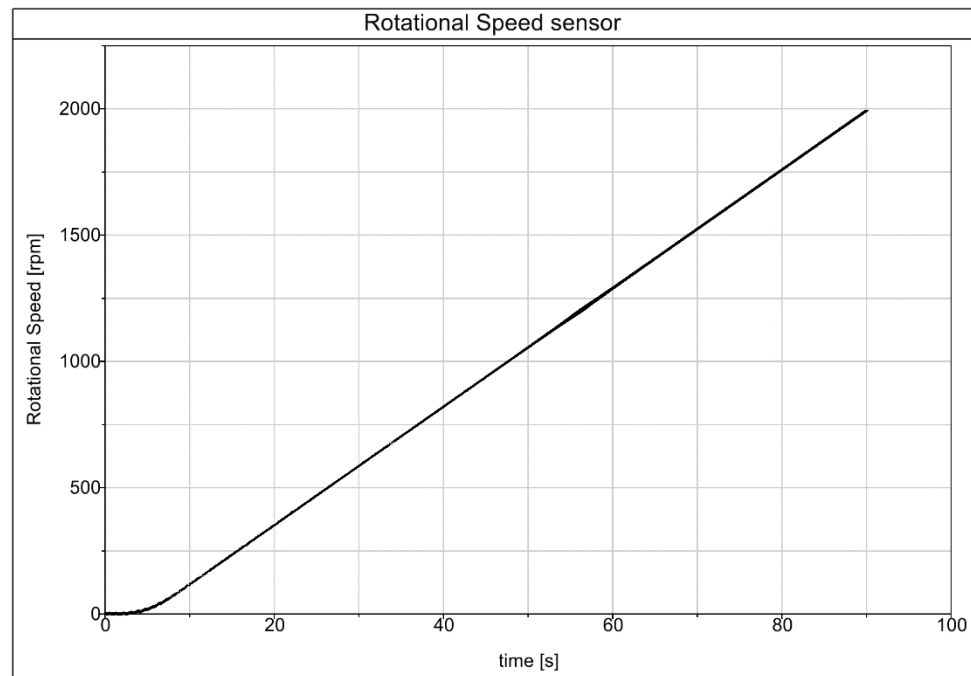
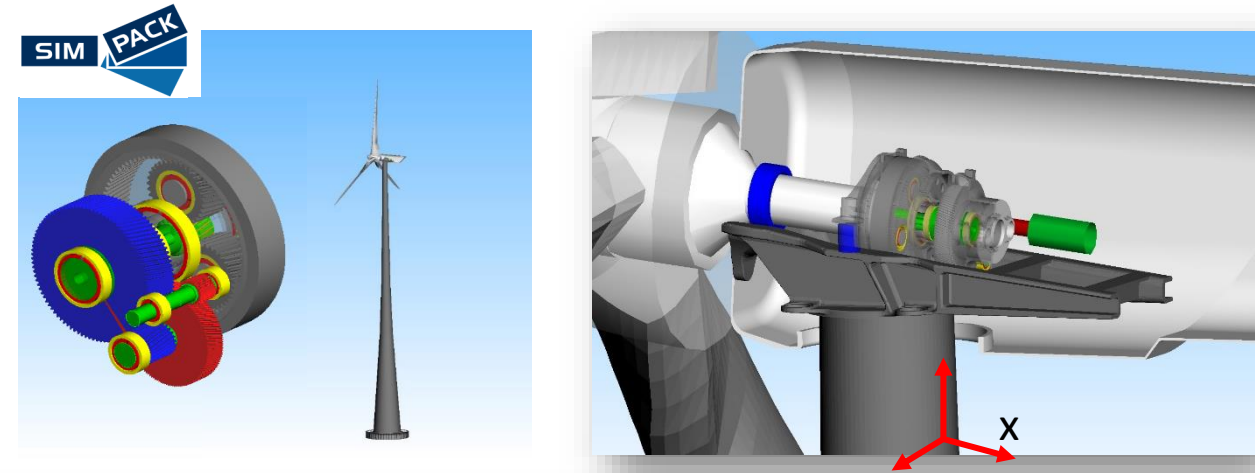
Vibration of radiating surfaces

- Audible noise



2 Identification of vibrations and their sources

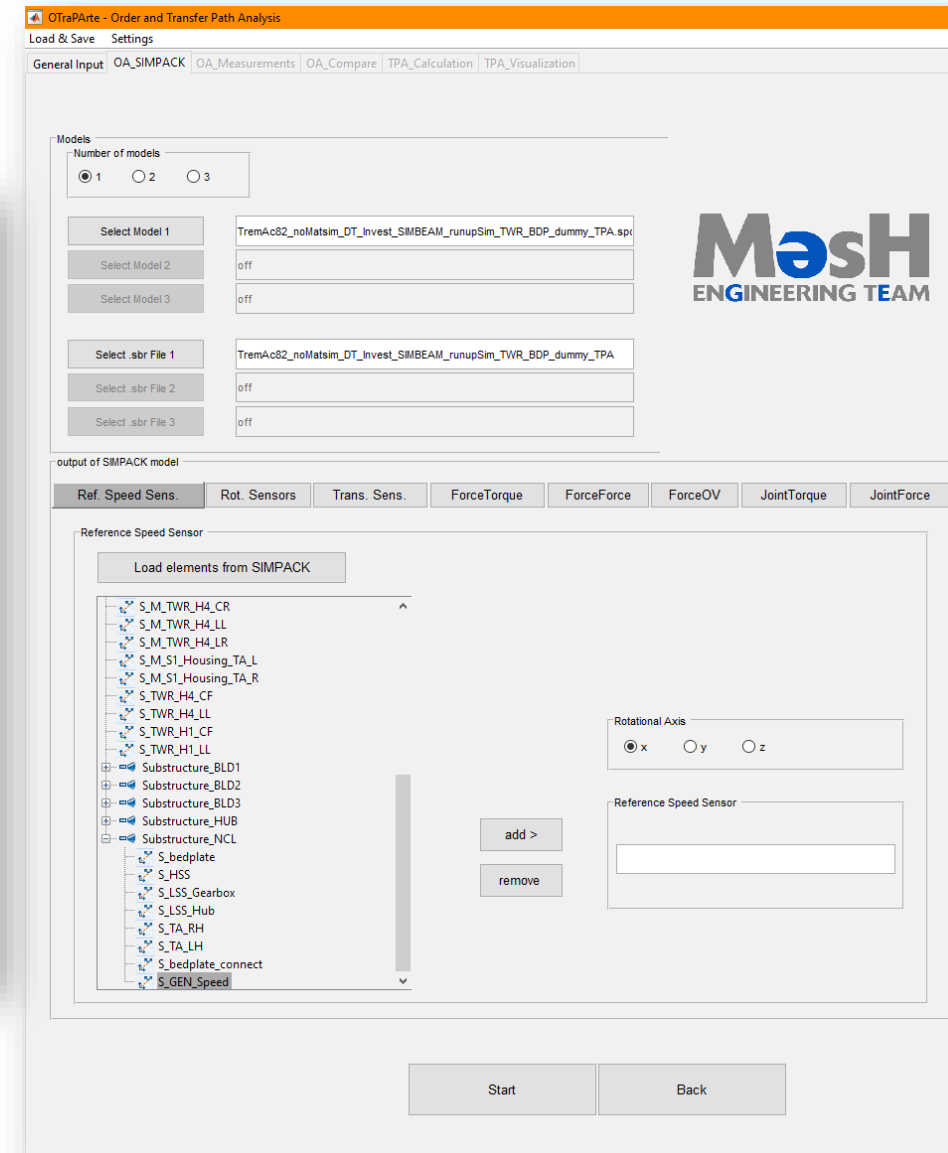
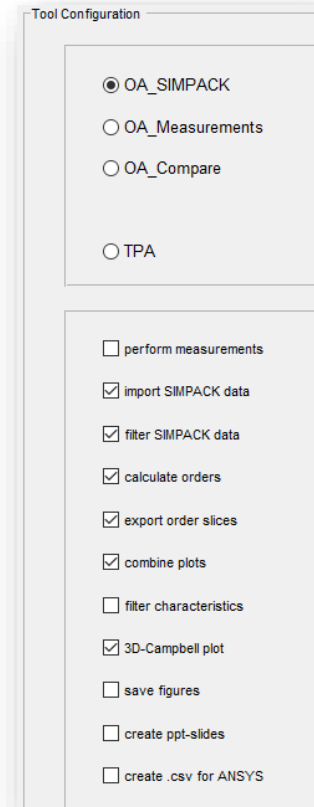
- **Simpack model of complete wind turbine**
 - Time-Domain Simulation
→ Run-Up in complete operating range
 - Gearbox consisting of three gear stages
 - Measurement of vibration with velocity sensors at tower surface



2 Identification of vibrations and their sources - OTraParTe

OTraParTe: Order Analysis features

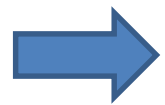
- Simpack Time Domain Results or physical measurement data
 - Comparison of up to 3 models
 - Selection of elements from Simpack model
 - Calling Simpack measurement
 - Filtering of data
 - FFT of data
 - Visualization & Export of 2D order diagrams and 3D-Campbell plots
 - Automatic creation of Power Point slides
 - Automatic creation of ANSYS-readable .csv files with interface loads



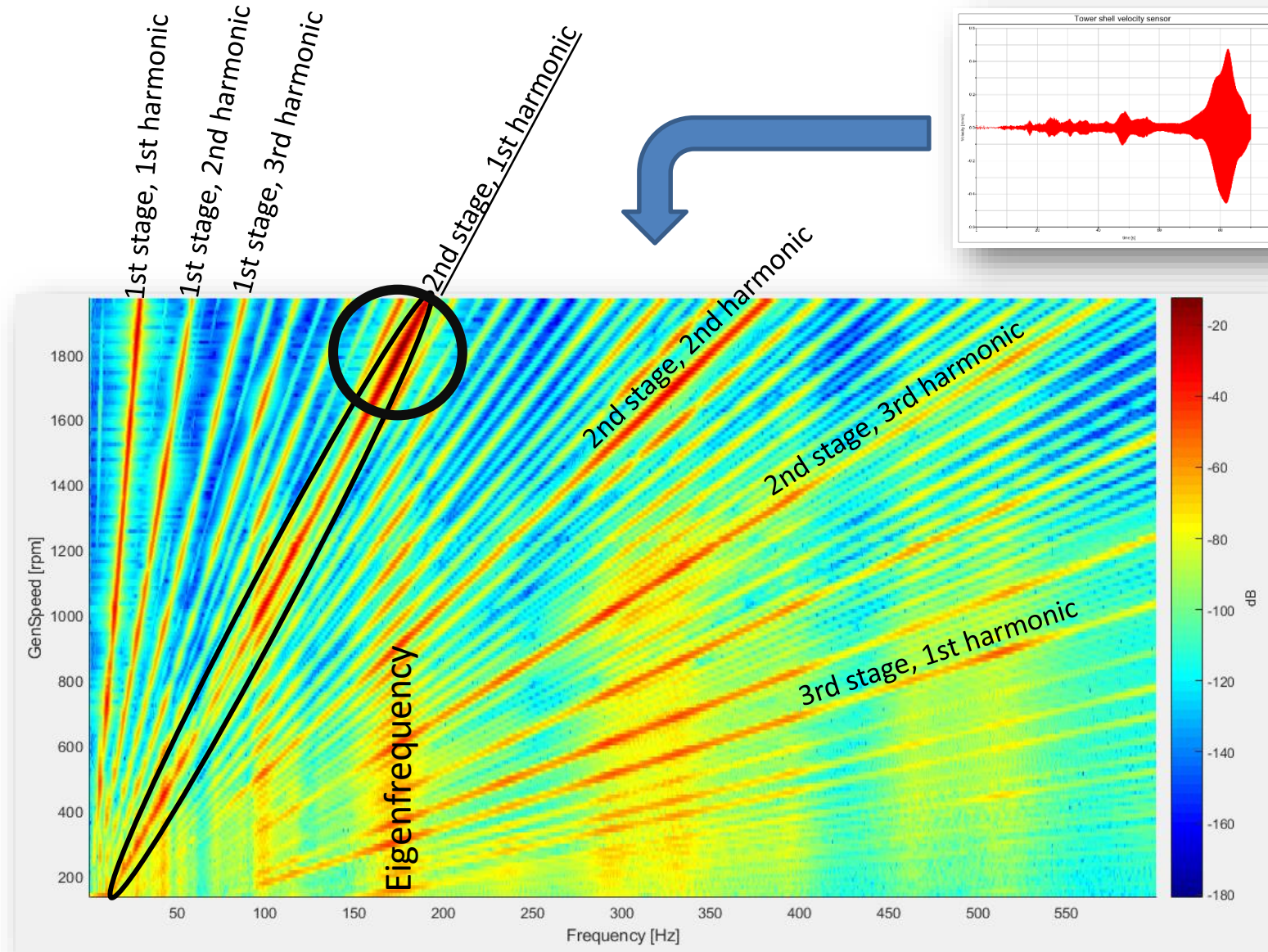
2 Identification of vibrations and their sources - OTraParTe

3D – Campbell Plot

- Frequency of response peaks is proportional to frequency of rotation
 - E.g. 2nd stage order: **5.8333**
- Skewed lines are the „order slices“
- For every gear stage there are multiple order slices → Harmonics
- Vertical distributions of amplitudes are eigenfrequencies
- Hot Spots are usually resonances → Avoid!



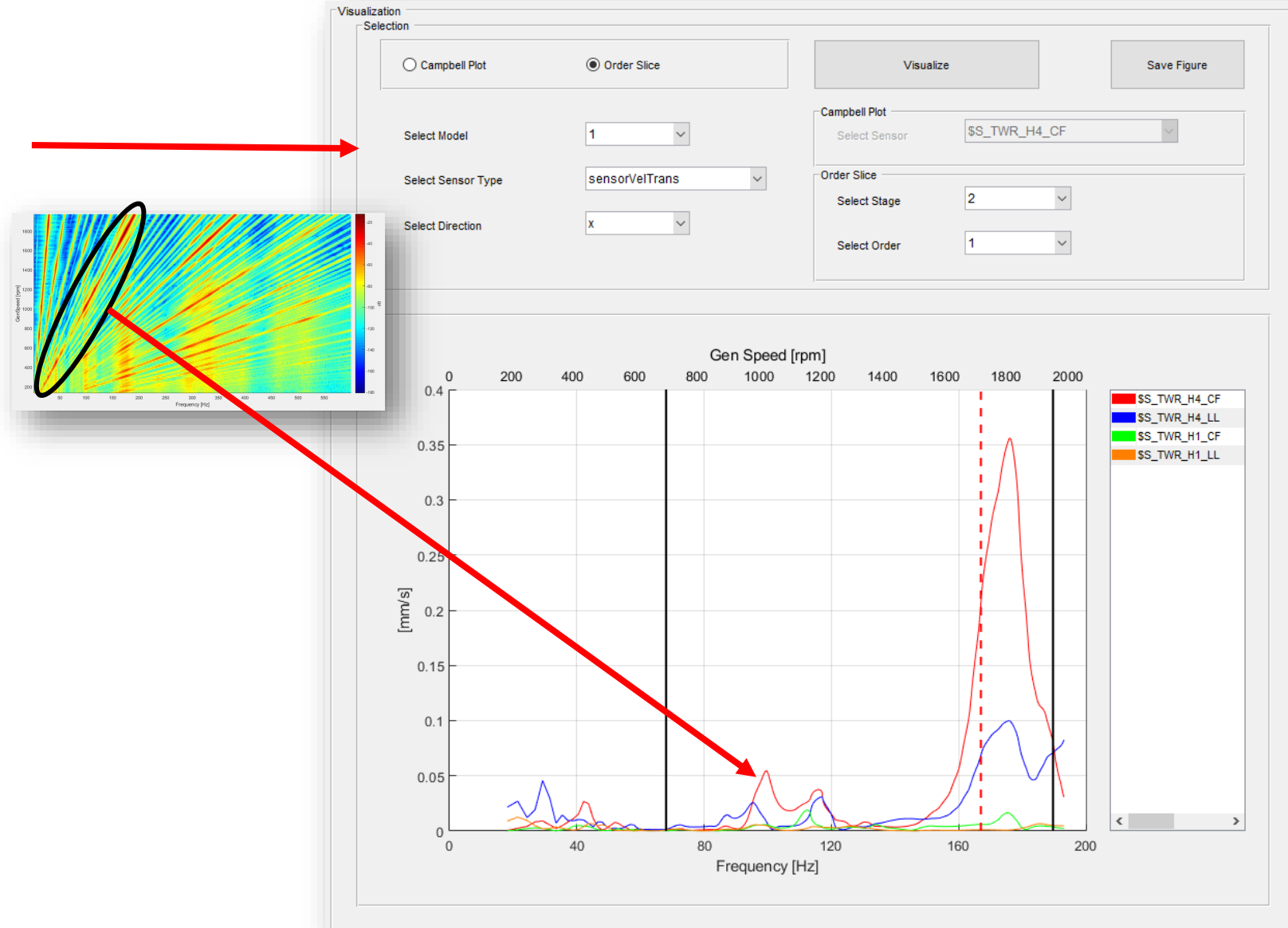
Resonant vibration identified on tower surface at 175Hz, excited by 2nd stage gear



2. Identification of vibrations and their sources - OTraParTe

2D – Order slice

- Selection of gearbox stage and order, sensor type and direction
- Extraction of sloped line from 3D -Campbell plot
- Comparison of multiple sensors
- Operating range and nominal speed also shown



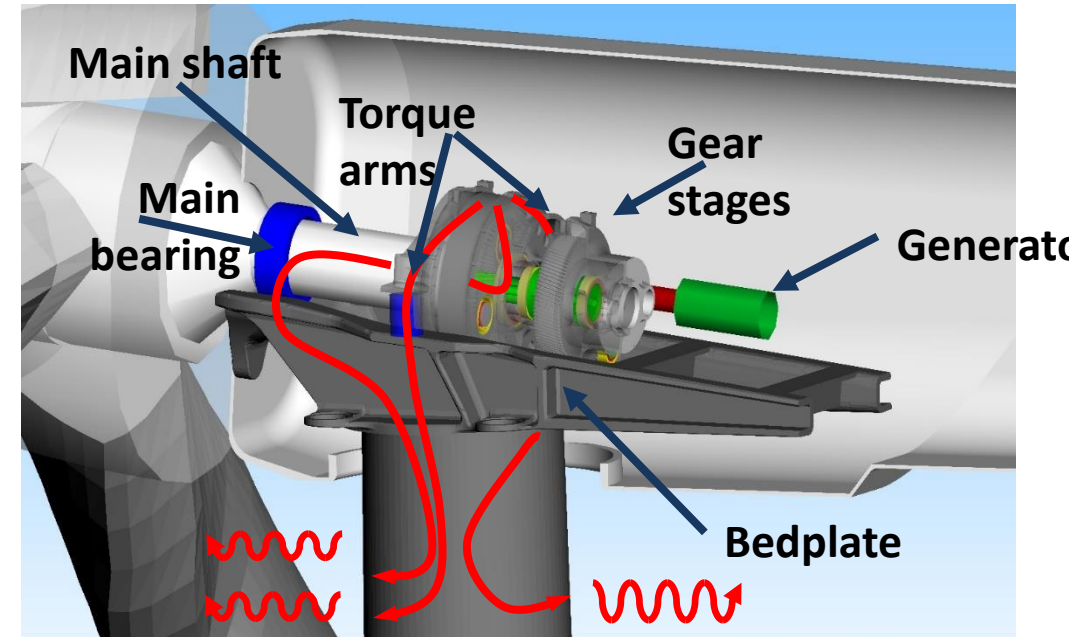
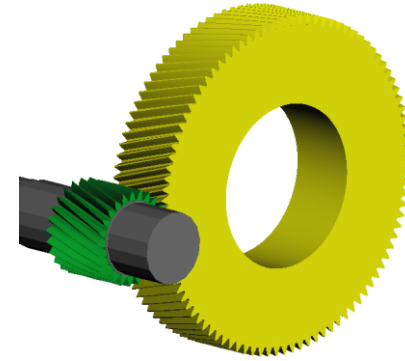
At a speed of 1800rpm we find the resonance with an amplitude of 0.35mm/s

2 Identification of vibrations and their sources

Source - Path - Receiver

- We found which „source“ is responsible for the resonant vibration
 - But what now?
- Minimizing or shifting the excitation...
 - Changing macrogeometry of gears
 - Changing microgeometry of gears... is always preferable
- But if you want to analyze the path a vibration takes to a receiver?

→ Transfer Path Analysis



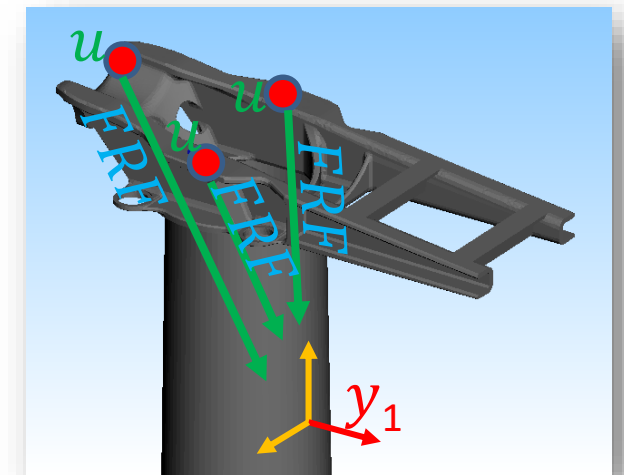
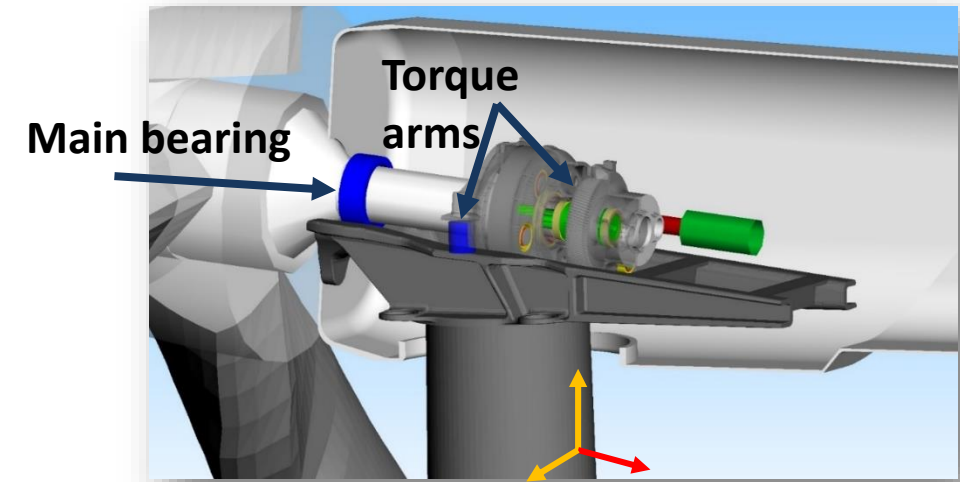
3 Understanding the transfer path of a vibration

- **What is a transfer path?**
 - Transfer path: Every possible connection
Source → Interface → Receiver
 - Definition of interfaces between the active parts (sources) and the passive parts
- **What do we need to do a Transfer Path Analysis?**
 - **u : Forces & Torques** transmitted at the interfaces
Here: 3 interfaces with 3 forces, 2 torques each
→ u has 15 components
 - The **FRF** (Frequency Response Functions) are obtained from Simpack Linear System Analysis

$$\begin{bmatrix} y_1 \\ y_2 \\ \vdots \\ y_n \end{bmatrix} = \begin{bmatrix} y_{11} + y_{12} + \dots + y_{1k} \\ y_{21} + y_{22} + \dots + y_{2k} \\ \vdots \\ y_{n1} + y_{n2} + \dots + y_{nk} \end{bmatrix} = \begin{bmatrix} FRF_{11} & FRF_{12} & \dots & FRF_{1k} \\ FRF_{21} & FRF_{22} & \dots & \vdots \\ \vdots & \vdots & \dots & \vdots \\ FRF_{n1} & \dots & \dots & FRF_{nk} \end{bmatrix} * \begin{bmatrix} u_1 \\ u_2 \\ \vdots \\ u_k \end{bmatrix}$$



- Components of the total response

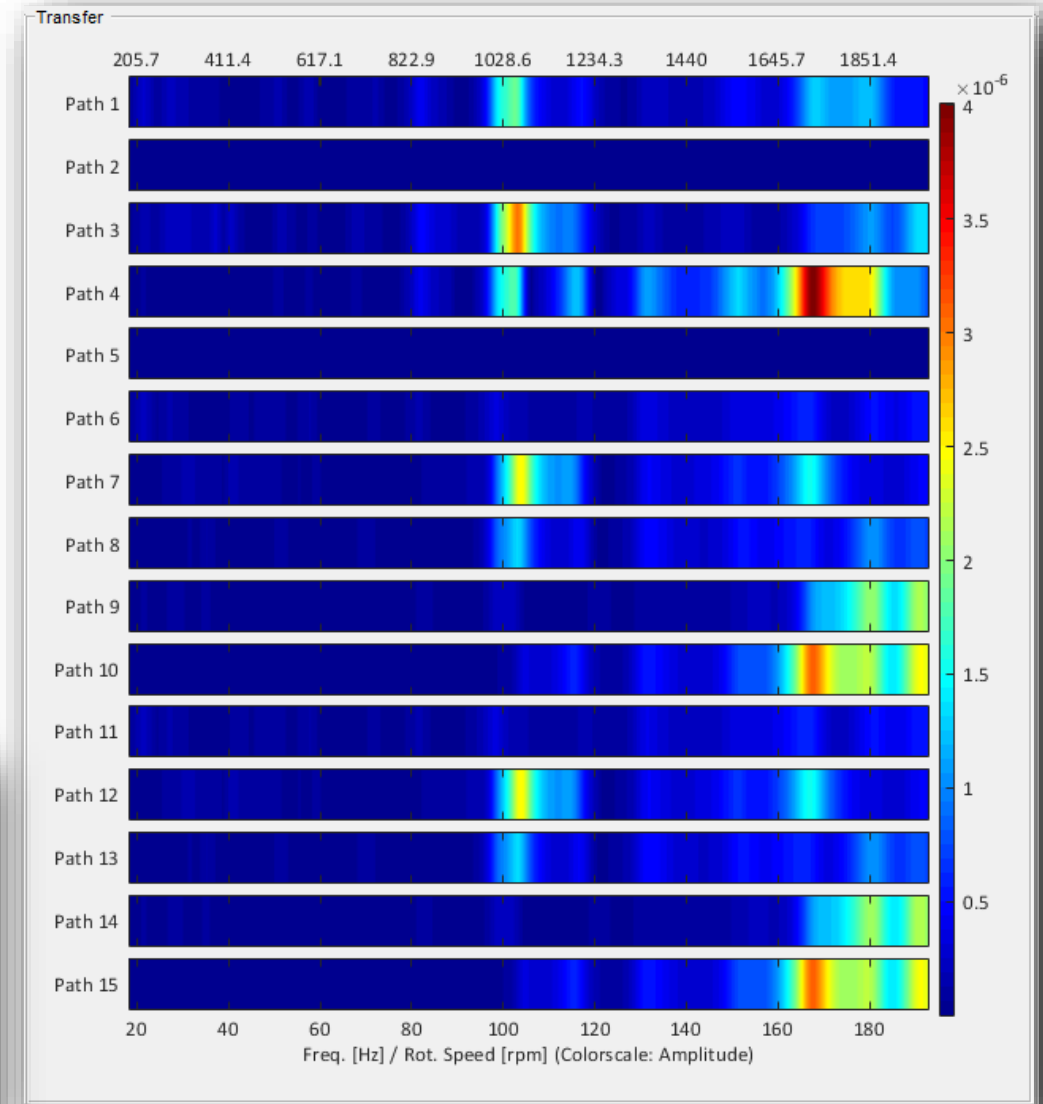
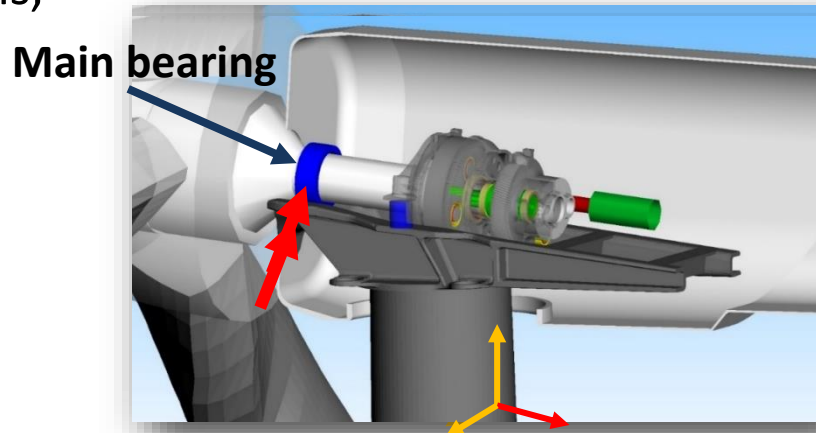


3 Understanding the transfer path of a vibration

$$\begin{bmatrix} y_1 \\ y_2 \\ \vdots \\ y_n \end{bmatrix} = \begin{bmatrix} y_{11} + y_{12} + \dots + y_{1k} \\ y_{21} + y_{22} + \dots + y_{2k} \\ \vdots \\ y_{n1} + y_{n2} + \dots + y_{nk} \end{bmatrix} = \begin{bmatrix} FRF_{11} & FRF_{12} & \dots & FRF_{1k} \\ FRF_{21} & FRF_{22} & \dots & \vdots \\ \vdots & \vdots & \dots & \vdots \\ FRF_{n1} & \dots & \dots & FRF_{nk} \end{bmatrix} * \begin{bmatrix} u_1 \\ u_2 \\ \vdots \\ u_k \end{bmatrix}$$

- **Spectrogram:** Overview over amplitudes of one order slice
- Path 4 is a dominant component! It is a torque at the main bearing, For each path, the FRF and interface load can be analyzed separately

→ At 175Hz, the structure is „sensitive“ to excitations,

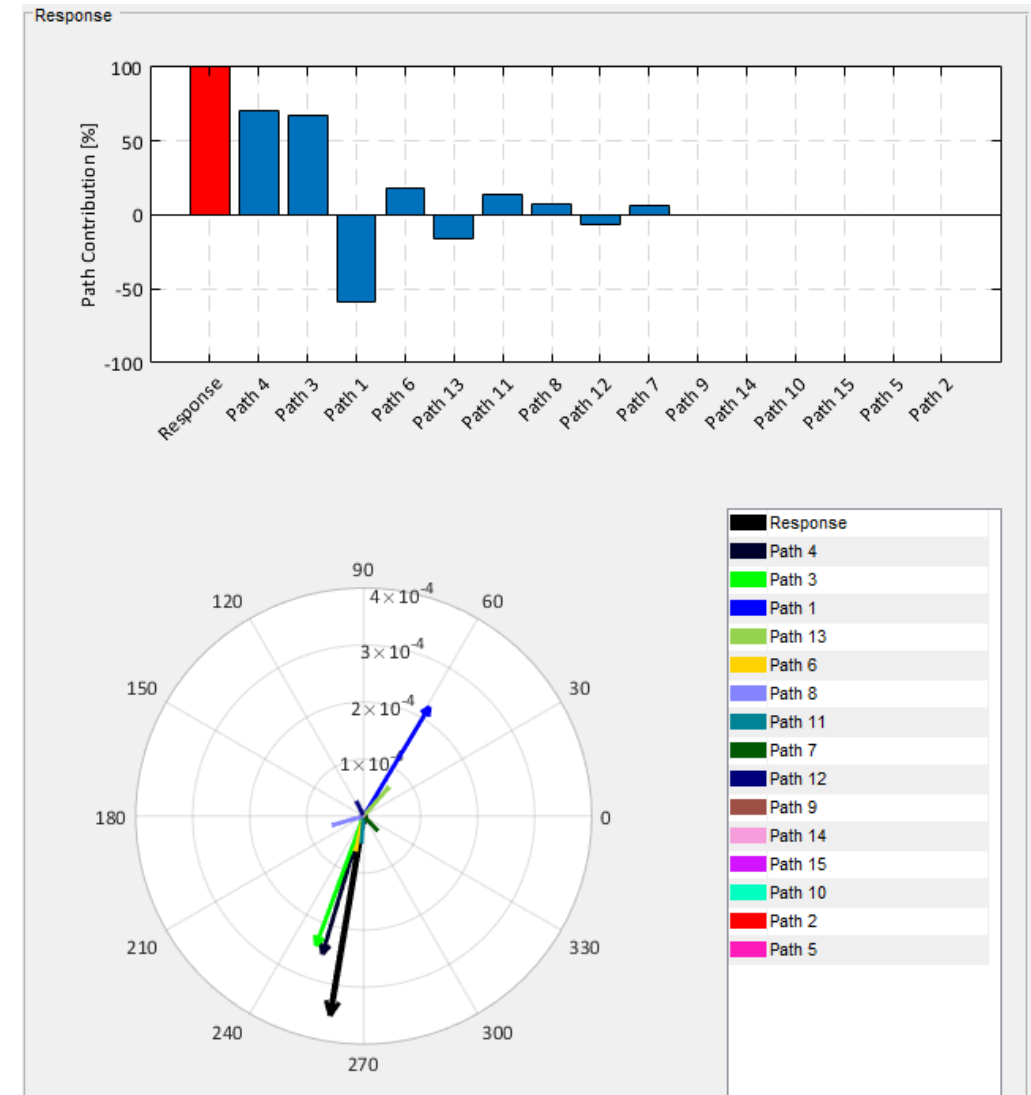


3 Understanding the transfer path of a vibration

$$\begin{bmatrix} y_1 \\ y_2 \\ \vdots \\ y_n \end{bmatrix} = \begin{bmatrix} y_{11} + y_{12} + \dots + y_{1k} \\ y_{21} + y_{22} + \dots + y_{2k} \\ \vdots \\ y_{n1} + y_{n2} + \dots + y_{nk} \end{bmatrix} = \begin{bmatrix} FRF_{11} & FRF_{12} & \dots & FRF_{1k} \\ FRF_{21} & FRF_{22} & \dots & \vdots \\ \vdots & \vdots & \dots & \vdots \\ FRF_{n1} & \dots & \dots & FRF_{nk} \end{bmatrix} * \begin{bmatrix} u_1 \\ u_2 \\ \vdots \\ u_k \end{bmatrix}$$

- Information of phase is important!
 - Response terms can cancel each other out
- Impact of phasing visible with Vector plots
- Vector plot of **response** for each path
- Vector plot of **interface load** for each path
- Vector plot of **FRF** for each path

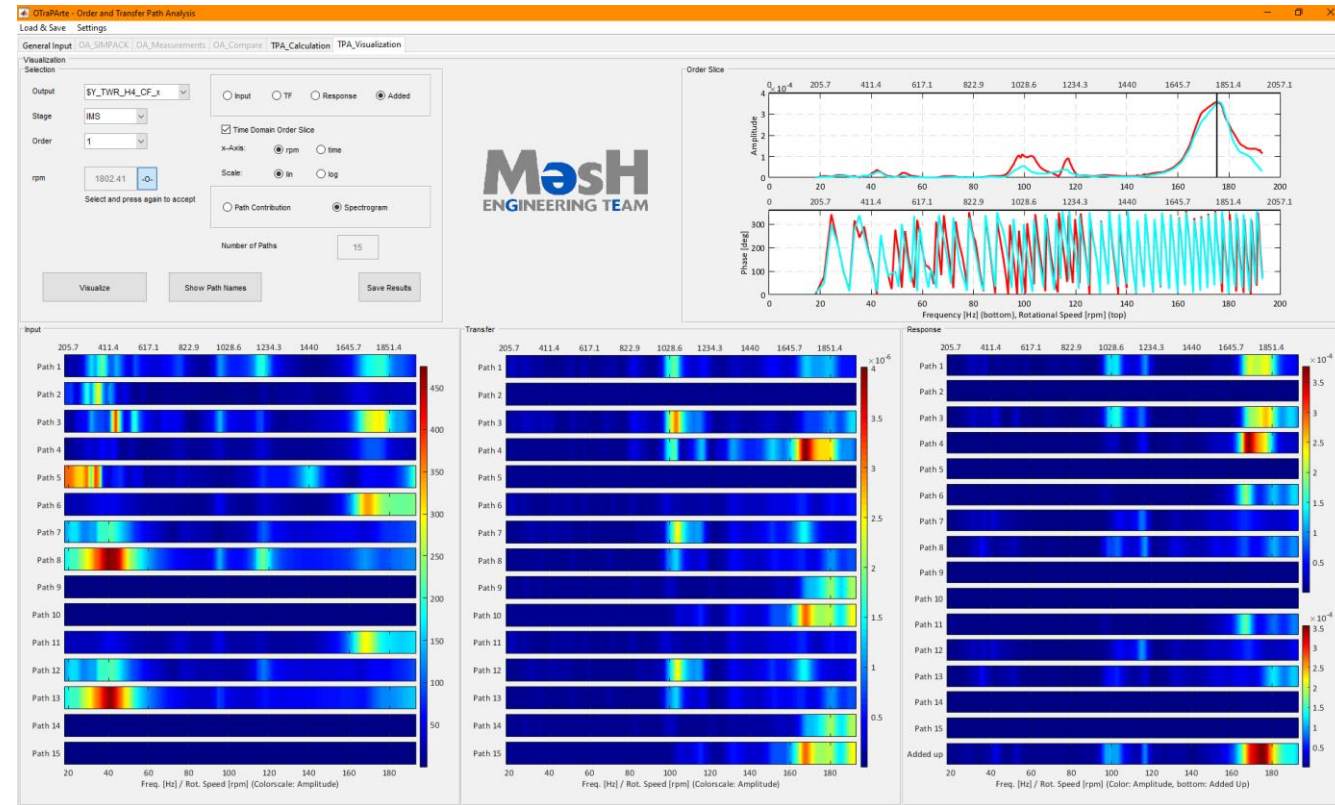
→ The response component of Path 1, is having an opposing phase to the main contributors Path 3 and Path 4

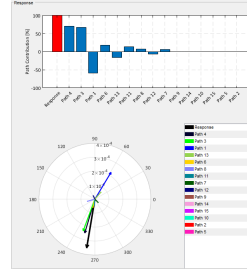
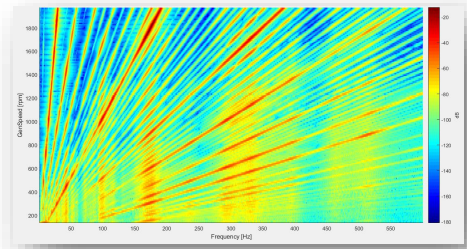


Conclusion

OTraPArTe – A GUI for:

- Order Analysis:
Identification of vibrations and their sources
- Transfer Path Analysis:
Understanding the transfer path of a vibration
- Usage possible in different engineering fields
- Combination of physical measurement data and Simpack results possible





Thank you for your kind attention!

Questions?

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wind@mesh-engineering.de